REMARKS

The above newly entered/amended paragraphs of the specification overcome some informalities noted in the specification. The undersigned avers that the newly entered/amended paragraphs of the specification do not contain any new subject matter.

Newly entered claims 5-8 merely rewrite the subject matter of original claims 1-4 in a more traditional U.S. claim format. The entered amendments are not, in any way, directed at distinguishing the present invention from any known prior art. Please consider the newly entered claims upon consideration of this application.

In the event that there are any fee deficiencies or additional fees are payable, please charge the same or credit any overpayment to our Deposit Account (Account No. 04-0213).

Respectfully submitted,

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REDLINE VERSION

[001]	HYDRAULICALLY HYDRAULIC SYSTEM	0 =
[002]	This application is a national stage completion of PCT/EP2004/002947 filed March 20, 2004 which claims priority from German Application Serial	0 •
	No.103 13 487.5 filed March 26, 2003.	0 =
[003]	FIELD OF THE INVENTION	0 =
[004]	The invention concerns a hydraulic system of the type defined in more detail	0 •
	in the preamble of claim 1.	0 •
[005]	BACKGROUND OF THE INVENTION	0 •
[006]	Hydraulic systems of this type actuate a consumer when pressure medium	
	from a pressure medium source is delivered into a piston space of a consumer in	
	order to displace a piston. The consumer can be, for example, a blockable	
	differential transmission in which the piston of the actuation device acts upon a	
	disk brake. In such a case, a decisive factor for driving comfort is the response	
	behavior of the differential block, which is determined by the speed of the piston.	
	Particularly when the outside temperature is low, the viscosity of the pressure liquid	
	increases and this considerably reduces the response of the piston and so too the	
	actuation speed of the differential block leading to undesired driving situations.	
[007]	DE 198 46 955 A1 discloses a reversing transmission that can be shifted	
	under load in which, to vent the piston space, when the actuator is not in its active	
	condition, pressure medium passes through the piston space via a throttle point	
	and a one-way valve and flows into the pressure reservoir. If the piston space is	
	pressurized, the one-way valve closes and pressure medium passes into the	
	piston space through a delivery line.	
[800]	The purpose of the present invention is to provide a hydraulic system in	
	which the actuation device can be operated rapidly and reliably even at low	
	temperatures.	
[009]	This objective is achieved by a hydraulic system of the type in question	0 =
	which also embodies the characterizing features of the principal claim.	0=

[010] SUMMARY OF THE INVENTION

[011]

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In the hydraulic system, pressure medium is drawn from a pressure medium reservoir by a pressure medium source, according to the invention; passes into the piston space, via a throttle point, in the non-actuated condition, and is then passed back into the pressure medium reservoir. This ensures that a stream of pressure medium is constantly flowing through the consumer and, therefore, that all the delivery lines are filled with pressure medium so that a uniform operating viscosity is maintained. When the piston space is pressurized and the consumer is actuated, the pressurization takes place via at least two pressure medium delivery lines and this ensures that the pressurization and the piston movement can take place very rapidly. Preferably, this is achieved, via a first pressure medium delivery line, connected by a 3/2-way valve to the pressure medium source and, via a second delivery line, which is permanently connected to the pressure medium source. If the hydraulic system comprises several consumers that have to be actuated at different pressure levels, a pressure-limiting valve can be fitted between the 3/2-way valve and the consumer to ensure that a defined pressure level is not exceeded at the consumer. This pressure-limiting valve can be continuously adjustable, for example, to set a defined slip condition of the blocking clutch at a differential.

[012] The hydraulic system, according to the invention, ensures that the piston of the consumer can be actuated with the same rapidity regardless of the external temperature. An additional advantage of the system is that it is of simple construction and comprises only a small number of components and valves.

[013]	BRIEF DESCRIPTION OF THE DRAWINGS	\$=
[014]	Other characteristics emerge from the description of the figure.	0=
[014]	The invention will now be described, by way of example, with reference to	0
	the accompanying drawings in which:	0-
[015]	The single Figure is a diagrammatic representation of a hydraulic system.	0-
		0 •
[016]	DETAILED DESCRIPTION OF THE INVENTION	0 •

[017] The single Figure represents a hydraulic system in which a pressure medium source 1 draws pressure medium from a pressure medium reservoir 2 and delivers it, on the one hand, to a 3/2-way valve 3 and, on the other hand, to a delivery line 4. When the 3/2-way valve 3 is in its starting condition, pressure medium passes through the line 4 to a throttle point 5 and from there into a second pressure medium deliver line 6 and through this into a piston space 7, from which the pressure medium can flow back via a first pressure medium delivery line 8 to the 3/2-way valve 3 and, from there, via a line 9 into the pressure medium reservoir 2. This ensures that when a consumer 10 is not actuated, pressure medium flows constantly from the pressure medium source 1 through the piston space 7 and from there back into the pressure medium reservoir 2. When the 3/2-way valve 3 is actuated, pressure medium passes via a delivery line 11 to the first pressure medium delivery line 8 and from there into the piston space 7 and, at the same time, pressure medium also flows via the throttle point 5 and the second pressure medium delivery line 6 into the piston space 7, whereby the consumer 10 is actuated rapidly and reliably. Consumers 12, 13 and 14 are controlled in an analogous manner. To be able to pressurize the consumers at different pressure levels, a pressure-limiting valve 15 is arranged between the 3/2-way valve 3 and the first pressure medium delivery line 8. Depending on its setting, the pressure in the piston space 7 can be controlled. The consumers 12, 13 and 14 can also be provided with corresponding pressurelimiting valves.

Reference numerals

- 1 Pressure medium source
- 2 Pressure medium reservoir
- 3 3/2-way valve
- 4 Delivery line
- 5 Throttle point
- 6 Second pressure medium delivery line
- 7 Piston space
- 8 First pressure medium delivery line
- 9 Delivery line
- 10 Consumer
- 11 Delivery line
- 12 Consumer
- 13 Consumer
- 14 Consumer
- 15 Pressure-limiting valve